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To cite this article: Morteza Mohammadzaheri, Amirhosein Amouzadeh, Mojtaba Doustmohammadi, Mohammadreza Emadi, Ehsan Jamshidi, Mojtaba Ghodsi & Payam Soltani (2021): Fuzzy Analysis of Resonance Frequencies for Structural Inspection of an Engine Cylinder Block, Fuzzy Information and Engineering, DOI: [10.1080/16168658.2021.1908819](https://doi.org/10.1080/16168658.2021.1908819)

To link to this article: <https://doi.org/10.1080/16168658.2021.1908819>



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Published online: 28 Jun 2021.



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Fuzzy Analysis of Resonance Frequencies for Structural Inspection of an Engine Cylinder Block

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ABSTRACT

Summary: A new inspection technique for complex mechanical structures is proposed in this paper, where a fuzzy inference system carries out structural inspection. The inputs to the fuzzy inference system are the elements of a fault signature, an array of numbers prepared with use of below 5 kHz resonance frequencies of faultless and a number of faulty specimens.

Advantage: Below 5 kHz resonance frequencies are easier and less expensive to obtain compared to higher frequency ones.

Limit: Due to high expenses of experiments, reliable finite element models were alternatively used to obtain resonance frequencies of the faulty specimens.

Results: The developed fuzzy inference system in this research accurately located an under-surface fault in an engine cylinder block.

ARTICLE HISTORY

Received 17 December 2020
Accepted 22 March 2021

KEYWORDS

Vibration; structural inspection; resonance frequencies; fuzzy; cylinder block

1. Introduction

Structural inspection [1], structural damage detection [2] or structural health monitoring [3] are roughly equivalent terms used for fault diagnosis of mechanical structures. Structural inspection ideally answers the following three questions: (i) Is there any faults? i.e. fault detection, (ii) where is (are) the fault(s)? i.e. fault isolation or localisation, (iii) how is the fault? (e.g. in terms of size and shape) i.e. fault identification [4–6]. Two approaches may be used in structural inspection or, in a wider view, fault diagnosis: (i) local or signal-based approach, and (ii) global or model-based approach.

In the first approach, the behaviour/information of the faultless system is obvious. For example, a metal part with no internal void space permits an ultrasonic wave to pass at a certain speed. Due to this evident response of the faultless system, only the response/information (in this example, the ultrasonic wave speed) of the faulty system is employed in fault diagnosis [7]. Such methods are called ‘signal-based’ methods [6, 8]. All signal-based techniques of structural inspection should be used in the locality of the

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fault [9]; hence, in the literature of structural inspection, *local* and *signal-based* approaches are equivalent [10]. Some other examples of local approach for structural inspection are radiography [11], CT scanning [12], magnetic field [13] and eddy-current [14] methods.

In global or model-based methods, a model or some behavioural information of the faultless system is needed for inspection. Vibration-based structural inspection methods belong to this approach [5, 15]. Vibration-based inspection techniques use modal/dynamic properties of the 'whole' system. In other words, vibration-based methods do not depend on the data collected from the fault locality; therefore, vibration-based methods are *global*. In general, terms 'global' and 'model-based' are practically exchangeable in the literature of structural inspection [5].

Vibration-based methods, as global methods for structural inspection, have the evident advantage of not being in need to access the locality of the fault. These methods can ideally provide information on damage existence (fault detection), location (fault isolation) and size (fault identification) for a mechanical structure [16]. NASA, in the late 1980s, employed vibration-based methods to inspect its shuttle instead of established signal-based methods [17]. However, some research disappointingly concluded that only higher frequency modes (e.g. with resonance frequencies over 30 kHz) are sensitive enough to local damages [9]; while, the measurement frequency of common acceleration sensors is up to 10 kHz; that is, modes with a resonance frequency up to 5 kHz can be captured with these sensors [18]. This conclusion would mean that only expensive vibration sensors with demanding operation could collect meaningful vibrational information for structural inspection. This disappointing conclusion faded the initial hopes to widespread use of vibration-based methods in quality control of manufactured metal parts [9].

However, this paper shows that well-developed fuzzy inference systems can successfully extract vivid structural inspection results out of low frequency vibrational information of a complex mechanical part, an engine block cylinder. Vibration-based structural inspection generally uses dynamic/modal properties of mechanical structures such as resonance frequencies, mode shapes and damping ratios [8, 16, 19–22]. This research employs the most easily obtainable dynamic properties, resonance frequencies at a fairly low frequency range (below 5 kHz) for structural inspection.

2. Problem Statement and Steps of Solution

This research aims to develop an algorithm including a fuzzy inference system (FIS). The inputs to the algorithm are below 5 kHz resonance frequencies of a faulty engine block cylinder (as a complex mechanical structure). The output of the algorithm is the fault location (extendible to other fault information in future research).

The investigated cylinder block, shown in Figure 1, belongs to a 1332 cm³ engine of Saba car made by Saipa company based in Iran. Some simplifications have been considered in this study: only a single fault of a particular type has been assumed to happen in the cylinder block, a spherical void with the diameter of 1 cm, located 2.5 cm above the bottom surface, a common size and depth for casting faults [23]. This removes four potential outputs of the fault diagnosis algorithm: number, shape, size and depth of the fault(s), only two potential outputs, two dimensions of the fault location, remain to be estimated. It is also assumed that a fault may only happen on a line. Thus, only a single dimension of the fault location remains as the sole output of the algorithm. However, as detailed in section 6 of the paper,

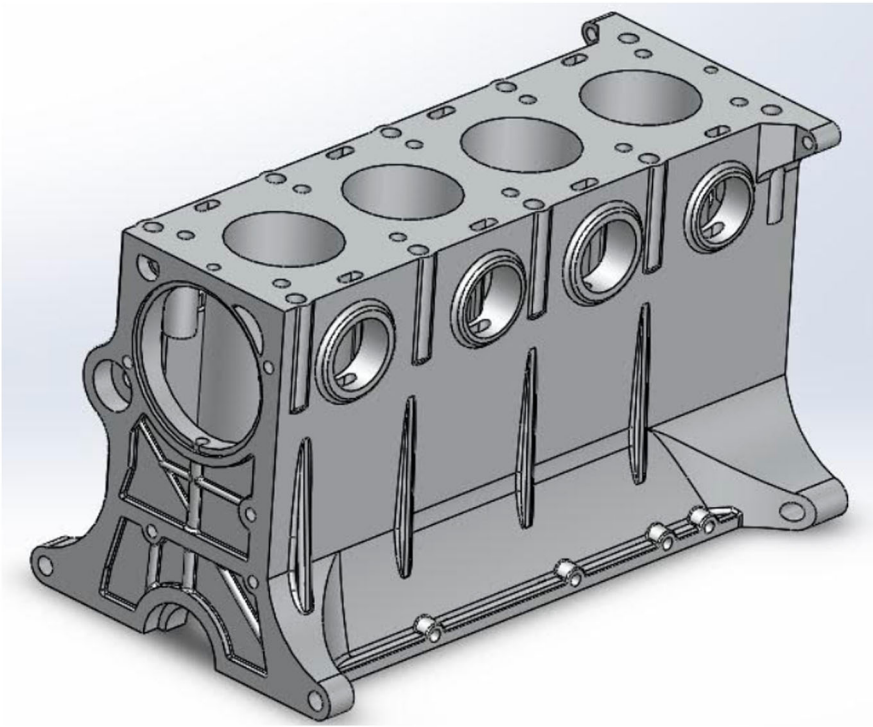


Figure 1. A computer model of the cylinder block.

successful development of an algorithm to identify one property of a fault in a complex structure paves the way to develop full scale structural inspection algorithms e.g. with six outputs.

3. Steps of Solution

Three following steps should be taken to develop the proposed fault isolation (or in general, structural inspection) algorithm:

- (1) Finding below 5 kHz resonance frequencies of the faultless specimen and a number of specimens with a fault, detailed in section 4.
- (2) Initial process of information collected at step 1 to produce a 'fault signature' associated with each fault location (in general, a signature for any faulty specimen). Section 5 details this step.
- (3) Development of a fuzzy inference system (FIS) to map the fault signatures, prepared in step 2, to the fault locations (in general, all fault information will be the destination of mapping). This step is presented in section 6.

4. Step 1 – Finding Resonance Frequencies

The resonance frequencies of the faultless specimen were found through experimental modal analysis, as shown in Figure 2. A DJB single-axis integrated-electronics piezoelectric

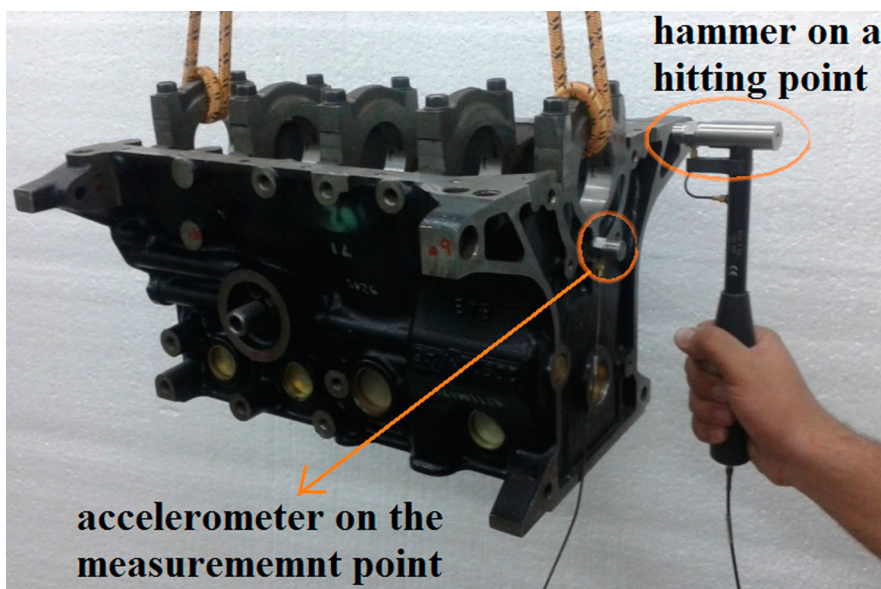


Figure 2. Upside down engine cylinder block during experimental modal analysis.

accelerometer of A/120/V type and an 8202 B&K impact hammer were used in experimental modal analysis; the acceleration of a single measurement point on the cylinder block was recorded, once the specimen was hit with a hammer on one of 21 other points. A B&K 3560 analyser was utilised to extract modal properties of the faultless cylinder block, including its resonance frequencies. Experimental modal analysis shows that the first 24 resonance frequencies of the faultless specimen are below 5 kHz.

Due to experimentation limits, it was impossible to have many faulty specimens and perform experimental analysis on each of them. Alternatively, a finite element model (FEM) of the faultless cylinder block was developed and experimentally validated. Then, for each fault location, a fault was added to the validated FEM on the specified location, and a FEM was developed for each faulty specimen. Then, resonance frequencies of faulty specimens were calculated through numerical modal analysis using their FEMs.

The FEM of the cylinder-block has 2002793 nodes and 1179381 irregular tetrahedral elements, constructed in ANSYS software package. In order to validate this model, the resonance (or modal) frequencies calculated out of numerical modal analysis of the FEM was compared to the ones obtained through experimental modal analysis, as presented in Table 1, where

$$\text{discrepancy}\% = \frac{|\text{experimental resonance frequency} - \text{FEM resonance frequency}|}{\text{experimental resonance frequency}} \times 100. \quad (1)$$

Twenty specimens (or FEMs) with a fault were used in this research. In Figure 3, circles roughly present fault locations. With considering the filled circle 2.5 cm far from the edge, as the origin, fault locations in cm can be listed as $x = [1 \ 2 \ 3 \ 4 \ 5 \ 6 \ 7 \ 8 \ 9 \ 10 \ 13.5 \ 14.5 \ 15.5 \ 16.5 \ 17.5 \ 18.5 \ 19.5 \ 20.5 \ 21.5 \ 22.5]$. Numerical modal analysis of faulty specimens' FEMs, for each of afore-listed fault locations, results in a list of 24 below 5 kHz resonance frequencies.

Table 1. Resonance frequencies resulting from experimental and FEM modal analysis.

Mode number	FEM resonance frequency (Hz)	Exp. resonance frequency (Hz)	Discrepancy %
1	1234.4	1239.7	0.4
2	1660.3	1653.6	0.4
3	2364.6	2375.9	0.5
4	2705	2698.4	0.2
5	3068.2	3071.4	0.1

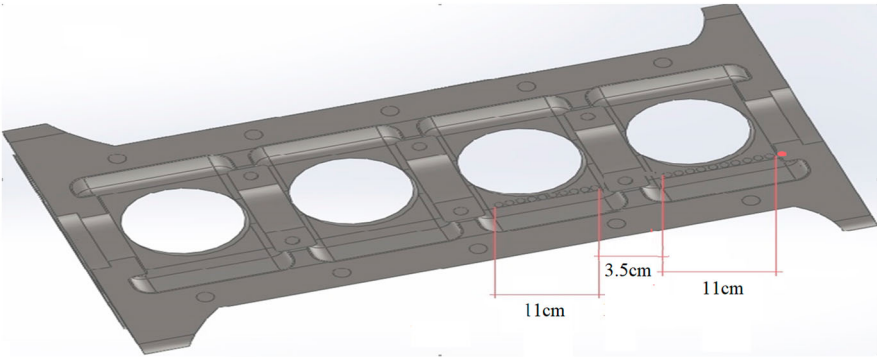


Figure 3. Fault locations on a section of the cylinder block. The filled circle, close to the edge is the origin.

5. Step 2 – Initial Process of Collected Resonance Frequencies

Experimental and numerical modal analysis, reported in section 4, result in 21 lists of 24 below 5 kHz resonance frequencies, one list for faultless part and 20 lists each associated with a fault location. The elements of the latter 20 lists are presented as ${}_k f_i$. i refers to the order of the vibrational mode associated with the resonance frequency, varying between 1 and 24; k refers to the fault location, varying between 1 and 20.

All ${}_k f_i$ s were deducted from the resonance frequencies of the faultless specimen associated with their own mode order f_i . The result is an array of 24 numbers for each fault location. This array is called the ‘signature of fault location’ or in short ‘fault signature’. ${}_k S_i$ in (2) is the i th element of the signature of k th fault location:

$${}_k S_i = {}_k f_i - f_i \quad (2)$$

S with $(20 \times 24 =)$ 480 elements, an array including 20 fault signatures with 24 elements each, is the output of the initial process of resonance frequencies obtained in step 1.

6. Step 3 – Development of Fuzzy Inference System

At this step, a FIS was developed to map the signature of a fault location \tilde{S} (as the input) to its corresponding location \tilde{x} (as the output). \tilde{x} and \tilde{S} may be ${}_k x$, k th fault location in x (presented in section 4) and its corresponding fault signature ${}_k S$, or they may be any another fault location (not listed in x , but within or near to the range of x) and its fault signature.

A linear Sugeno-type FIS was used in this research [24]. These fuzzy systems have been successfully used to tackle engineering problems in the areas of data-driven modelling

[25], estimation [26] and virtual sensing [27] and can be converted into neuro-fuzzy networks [28].

Let us assume that the FIS has n rules, where n is yet to be identified. Each rule receives all 24 elements of \tilde{S} as inputs and has a membership function per input. The output of each membership function is a membership grade. In this research, a Gaussian membership function of (3) produces a membership grade, μ_{ij} , for j th rule and i th input (\tilde{S}_i),

$$\mu_{ij} = \exp \left(-\frac{(\tilde{S}_i - C_{ij})^2}{2\Omega_{ij}^2} \right). \quad (3)$$

where C_{ij} and Ω_{ij} are the centre and the width of the membership function, respectively. The product of membership grades of a rule was considered as the weight of the rule, a number between zero and one, appearing in the denominator of (4). In addition, in linear Sugeno fuzzy systems, the output of each rule is a linear combination of its inputs (\tilde{S}_i s, elements of the signature of a fault location), as shown in the numerator of (4). The output of the whole model (\hat{x} , the fault location) is the weighted sum of rules outputs:

$$\hat{x} = \frac{\sum_{j=1}^n \left(\overbrace{\left(\sum_{i=1}^{24} A_{ij} \tilde{S}_i + B_j \right)}^{j^{th} \text{ rule output}} \prod_{i=1}^{24} \mu_{ij} \right)}{\sum_{j=1}^n \underbrace{\prod_{i=1}^{24} \mu_{ij}}_{j^{th} \text{ rule weight}}}. \quad (4)$$

where $\hat{}$ stands for estimated. Model (4), if fully developed and validated, can estimate the fault location (\hat{x}) with use of fault signature (\tilde{S}). However, for this purpose, the values of n (number of rules) and the elements of A , B , C and Ω should be identified.

Development of the FIS requires two steps:

- Model generation: finding the number of rules, n , and initial estimation of model parameters or the elements of A , B , C and Ω
- Parameter identification: determining model parameters accurately. The developed FIS should be cross-validated afterwards.

In this research, the FIS was developed with use of a data-driven approach. Therefore, the data prepared in section 5 were utilised, composed of a list of 20 fault locations, x , and their associated fault signatures. Three sets of data were needed for this research, *modelling data* (to be used in both model generation and parameter identification), *validation data* (to prevent overfitting in parameter identification) and *test data* to cross-validate the developed FIS. The fault location of 18.5 cm and its fault signatures form the test data; while the validation data includes the fault locations of 7, 10 and 20.5 cm and their fault signatures. The rest of the prepared data are the modelling data.

Model generation was carried out with modelling data using subtractive clustering technique, as detailed in [29], with the following coefficients: Range of Influence = 0.5, Squash Factor = 1.25, Accept Ratio = 0.1 and Reject Ratio = 0.15.

For parameter identification, first, the 'modelling error', E_m , was defined to represent the discrepancy of real fault location and the one estimated by the FIS (with $\hat{\cdot}$) for the modelling data. (5) shows the general formula to find error for modelling (E_m), validation (E_v) or tests data (E_t).

$$E = \frac{\sum_{\text{for a data set}} ({}_k\hat{x} - {}_kx)^2}{\text{number of fault locations in the data set}}. \quad (5)$$

The parameters of the model were adjusted using an iterative algorithm [29] so as to minimise the modelling error. First, the least square of error technique [30] adjusts the elements of A and B of the initial model (the output of subtractive clustering). Then, error backpropagation with gradient decent method with a variable step size [31] adjusts C and Ω elements. Afterwards, E_v was calculated. These three steps were performed iteratively, till the validation error, E_v , at one iteration exceeds E_v of its previous iteration. This situation, $E_v > \text{previous } E_v$, is a sign of overfitting and signals to stop the iterative algorithm of parameter identification [25]. If the algorithm did not stop in the case of overfitting, the modelling error would further decrease at the cost of loss in generality of the FIS [32]. The resultant FIS still needs to be cross-validated. A widely accepted cross-validation criterion is that the estimation output by the model calculated with the test data (never used in parameter identification directly or indirectly) should be acceptable [33, 34]; that is, E_t should be small enough. Here is a pseudocode for the development of the FIS:

- 10 Initial modelling through subtractive clustering with use of modelling data (n is identified)
- Calculate E_m and E_v
- *While E_v is decreasing*
 - Adjust A and B elements so as to decrease E_m through the method of least square of errors
 - Adjust C and Ω elements so as to decrease E_m through backpropagation gradient descent method with variable step size
 - Determine E_v and E_m
- Calculate E_t (test error)
- If E_t is unacceptable go to 10

7. Results and Discussion

In this research, subtractive clustering led to an FIS with 16 rules. The developed FIS estimates the location of 18.5 cm as 17.47 cm. It is an acceptable result considering that the data of this fault location have not been used in FIS development at all. That is, the FIS is cross-validated in the operating area in which its modelling, validation and test data have been collected.

The FIS has been developed using below 5 kHz resonance frequencies. That is, easy to collect vibrational data can be effectively used for fault isolation in complex engine parts.

Considering (3) and (4), and $n = 16$, A , C and Ω each has $(24 \times 16 =)384$ elements each, and B has 16 elements. Therefore, the FIS has 1168 parameters altogether, considerably more than the elements of all data arrays, which are 500, composed of 480 elements of S and 20 fault locations. This shows that the number of experiments or reliable numerical simulations are fairly small, and modal analysis of a larger number of faulty specimens could result in a much higher accuracy. In the case of availability of results from experimental or reliable numerical modal analyses, it is theoretically possible to update the aforementioned pseudocode/formulae to increase the number of FIS outputs or use parallel single output FISs to estimate two or three dimensions of the fault location or even size and shape factors of the fault(s). Six parallel FISs, similar to the one developed in this research, would roughly have 7008 parameters; one can expect that thousands of modal analyses with different fault locations/sizes/numbers/shapes would be needed to develop an FIS (or a combination of single output FISs) to carry out structural inspection in full. In total, the proposed method requires the results of several modal analyses prior to FIS development; this seems to be the major drawback of the method.

8. Conclusion

This paper presents a new method to structural inspection of complex mechanical parts, based on development of a fuzzy inference system. The proposed method uses resonance frequencies of below 5 kHz, which are much easier to obtain compared to the ones at higher frequency ranges or other modal properties. Development of the fuzzy inference system requires three steps, first, finding below 5 kHz resonance frequencies both for faultless specimen and for a number of faulty specimens. In this research, a fault was added to a validated FEM, on a different location per FEM, and then numerical modal analysis was carried out to find resonance frequencies. The second step is to process the obtained resonance frequencies to form a signature (an array of numbers) for each fault location. Afterwards, at the third and last step, a fuzzy inference was developed to map signatures to their associated fault locations. The developed fuzzy inference system in this research could locate an under-surface fault accurately in an engine cylinder block.

Disclosure statement

No potential conflict of interest was reported by the author(s).

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